

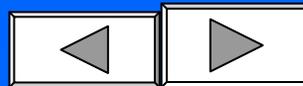
# 有限元教学案例

## 有电



杨庆生  
制作

之三



# 本课件包括五部分:

## 一、绪论

### 第一章 绪论

## 二、弹性力学基础

### 第二章 基本概念与假设

### 第三章 平面问题的基本理论

## 三、有限元理论及程序

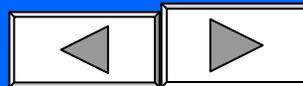
### 第四章 有限元法的基本概念

### 第五章 等参元

### 第六章 三角形单元计算机程序

## 四、有限元的扩展

## 五、应用



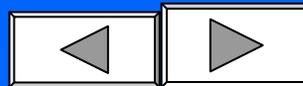
# 三、有限元法的理论及程序

(16课时)

第四章 有限元法的基本概念

第五章 等参元

第六章 三角形单元计算机程序



# 第四章 有限元法的基本概念

4.1 弹性力学问题的离散化

4.2 有限元的分析过程

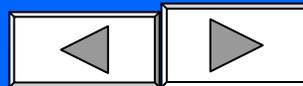
4.3 单元分析

4.4 解答的收敛性准则

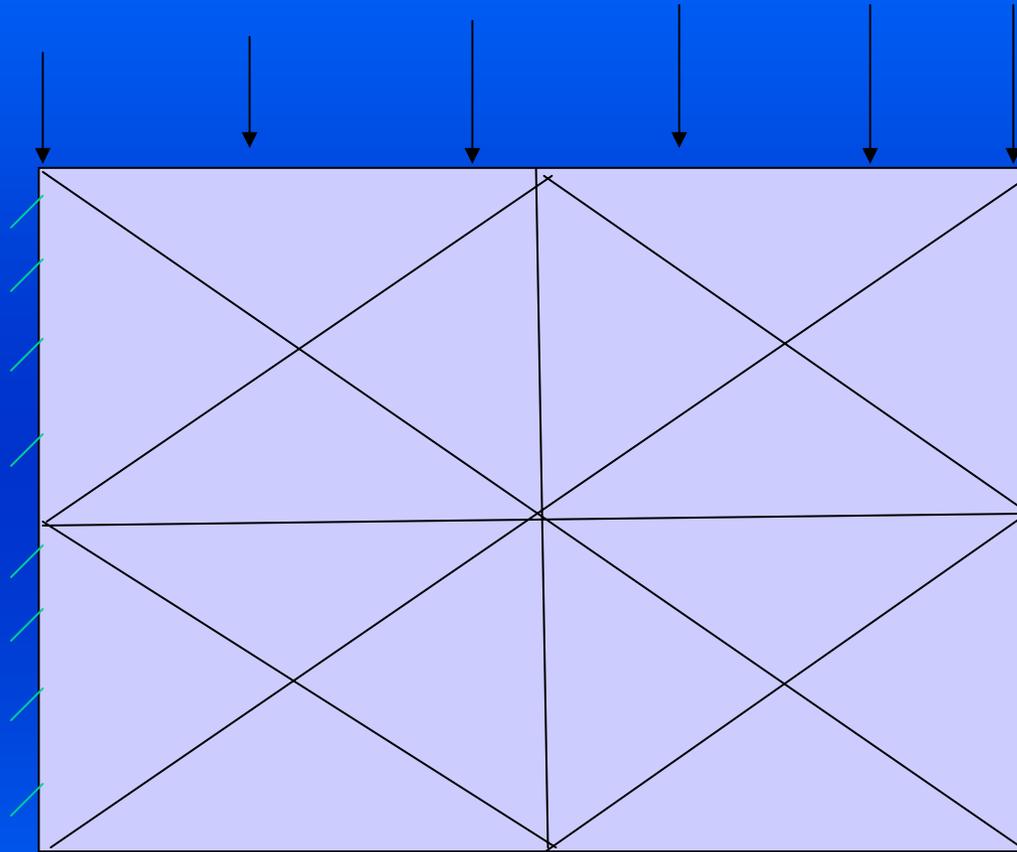
4.5 单元刚度阵的另一推导

4.6 整体分析

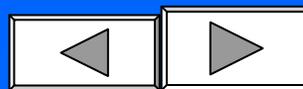
4.7 边界条件的引入

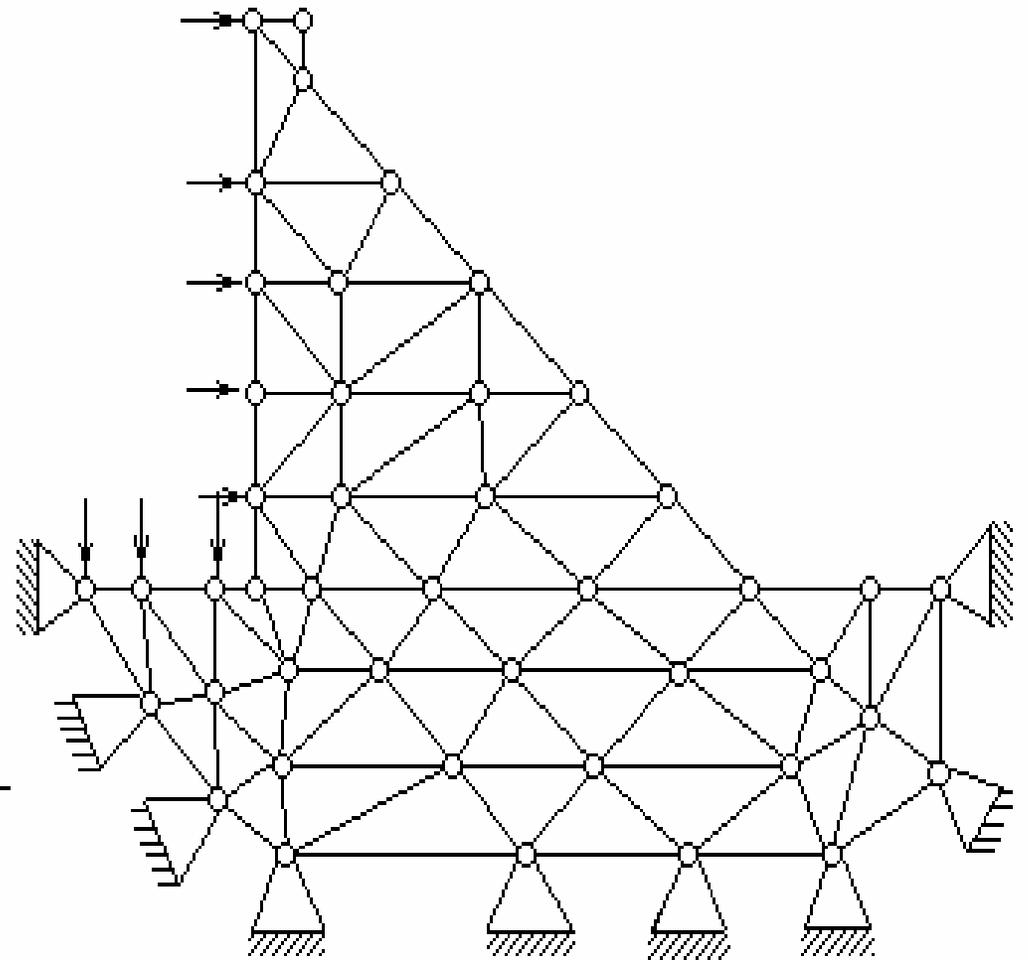
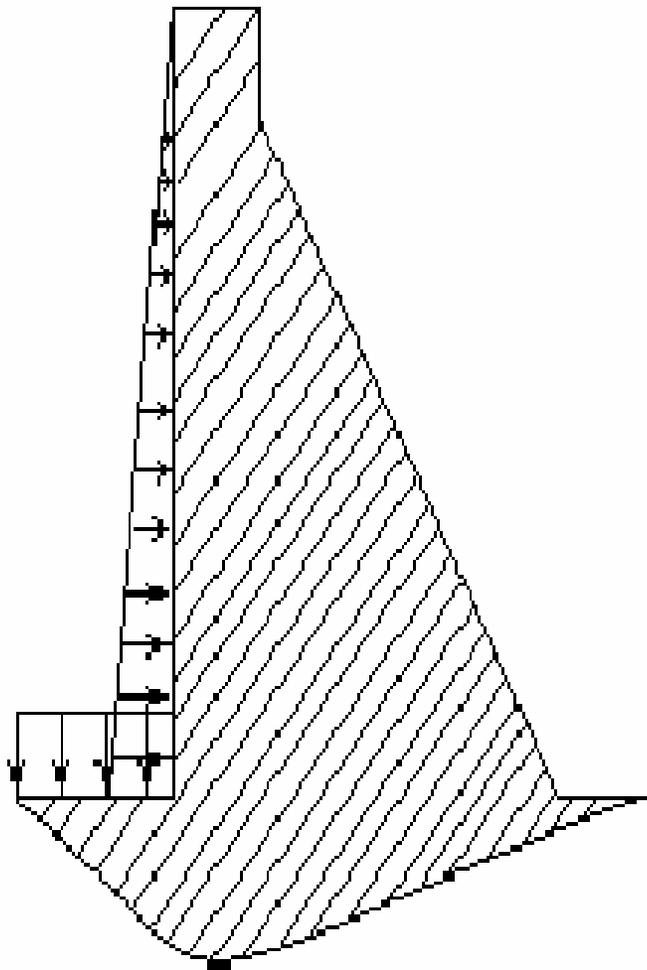


## 4.1 弹性力学问题的离散化

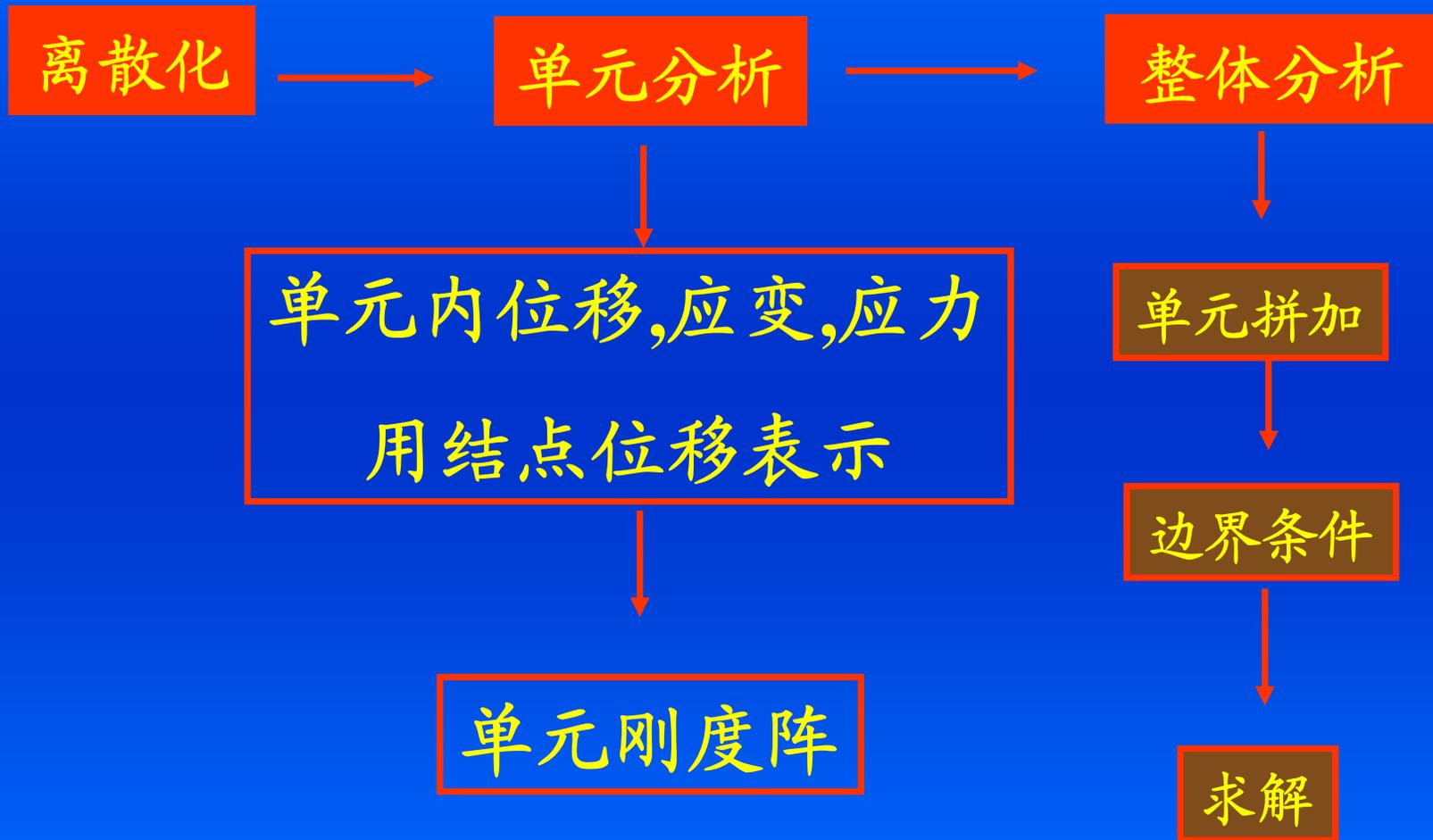


单元, 结点





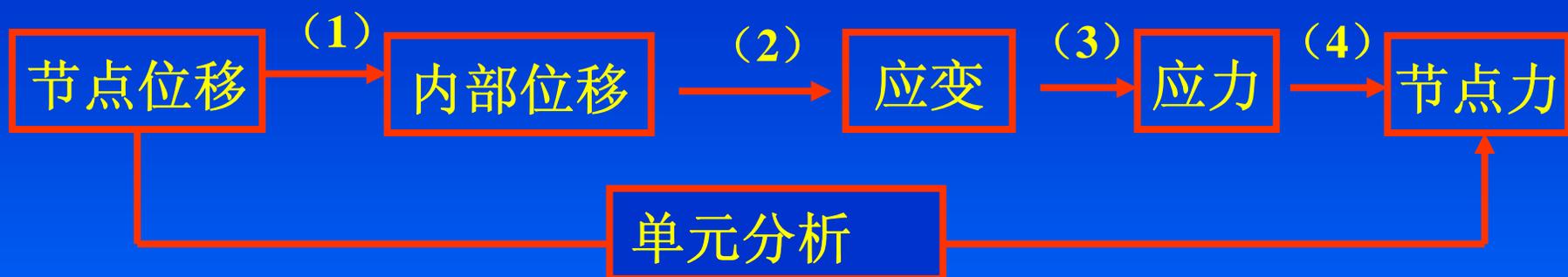
## 4.2 有限元的分析过程

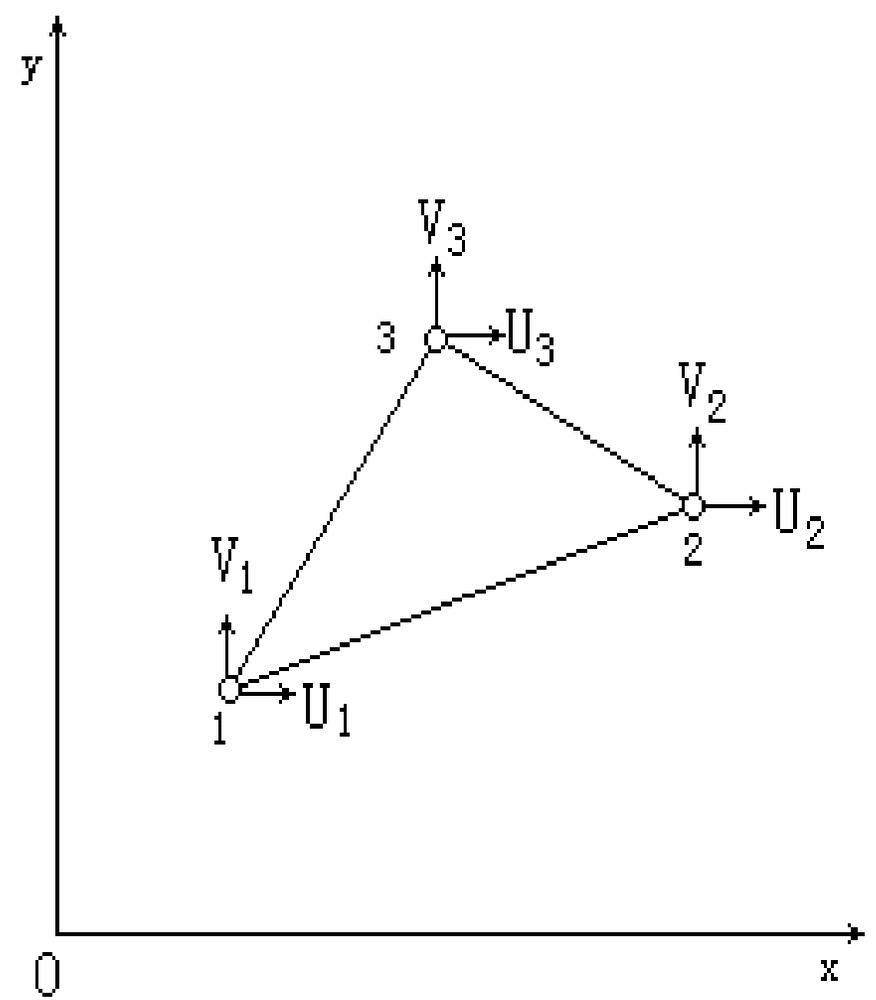
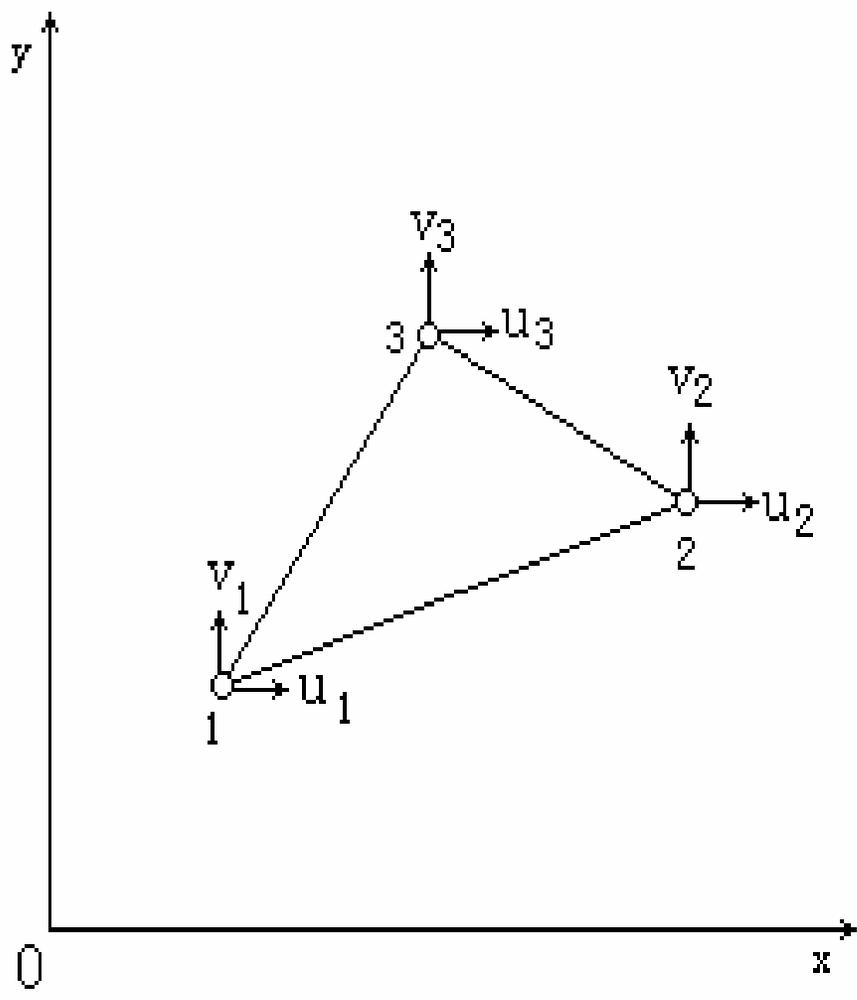


## 4.3 单元分析

### 常应变三角形单元

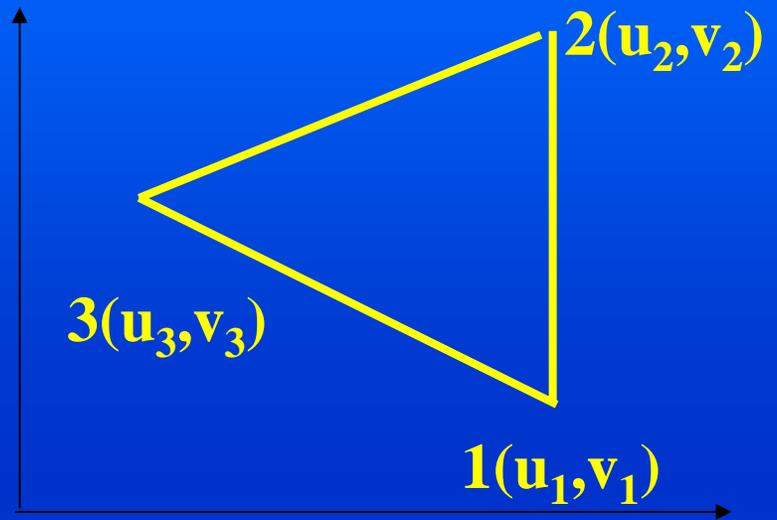
单元分析的步骤如下：





## 1. 结点位移(结点变量)

$$\{a_1\} = \begin{Bmatrix} u_1 \\ v_1 \end{Bmatrix}$$



$$\{a\} = \begin{Bmatrix} \{a_1\} \\ \{a_2\} \\ \{a_3\} \end{Bmatrix} = \begin{Bmatrix} u_1 \\ v_1 \\ \vdots \\ v_3 \end{Bmatrix}$$

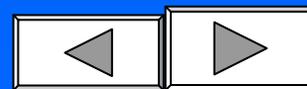
## 结点力

$$\{F\} = \begin{Bmatrix} \{F_1\} \\ \{F_2\} \\ \{F_3\} \end{Bmatrix} = \begin{Bmatrix} U_1 \\ V_1 \\ \vdots \\ V_3 \end{Bmatrix}$$

## 单元位移模式

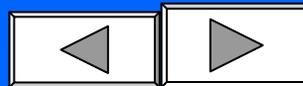
单元内任何一点的位移用结点位移表示

$$\begin{cases} u(x, y) = \beta_1 + \beta_2 x + \beta_3 y \\ v(x, y) = \beta_4 + \beta_5 x + \beta_6 y \end{cases}$$



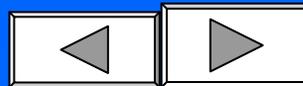
$$\{d\} = \begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} 1 & x & y & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & x & y \end{bmatrix} \begin{Bmatrix} \beta_1 \\ \beta_2 \\ \beta_3 \\ \beta_4 \\ \beta_5 \\ \beta_6 \end{Bmatrix}$$

$$\begin{cases} u_1 = \beta_1 + \beta_2 x_1 + \beta_3 y_1 \\ u_2 = \beta_1 + \beta_2 x_2 + \beta_3 y_2 \\ u_3 = \beta_1 + \beta_2 x_3 + \beta_3 y_3 \end{cases}$$



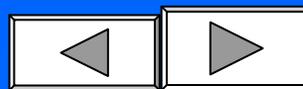
$$\begin{cases} v_1 = \beta_4 + \beta_5 x_1 + \beta_6 y_1 \\ v_2 = \beta_4 + \beta_5 x_2 + \beta_6 y_2 \\ v_3 = \beta_4 + \beta_5 x_3 + \beta_6 y_3 \end{cases}$$

$$\begin{Bmatrix} \beta_1 \\ \beta_2 \\ \beta_3 \\ \beta_4 \\ \beta_5 \\ \beta_6 \end{Bmatrix} = \frac{1}{2\Delta} \begin{bmatrix} a_1 & 0 & a_2 & 0 & a_3 & 0 \\ b_1 & 0 & b_2 & 0 & b_3 & 0 \\ c_1 & 0 & c_2 & 0 & c_3 & 0 \\ 0 & a_1 & 0 & 0 & 0 & c_3 \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ v_2 \\ v_3 \end{Bmatrix}$$



$$\{d\} = \begin{Bmatrix} u \\ v \end{Bmatrix} =$$

$$\begin{bmatrix} 1 & x & y & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & x & y \end{bmatrix} \frac{1}{2\Delta} \begin{bmatrix} a_1 & 0 & a_2 & 0 & a_3 & 0 \\ b_1 & 0 & b_2 & 0 & b_3 & 0 \\ c_1 & 0 & c_2 & 0 & c_3 & 0 \\ 0 & a_1 & & & & \\ & & & & & \\ & & & & & c_3 \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ \\ \\ \\ v_3 \end{Bmatrix}$$



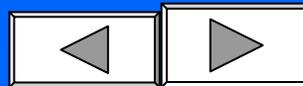
$$\{d\} = \begin{Bmatrix} u \\ v \end{Bmatrix} = [N(x, y)]\{a\}$$

$$[N(x, y)] = \begin{bmatrix} N_1 & 0 & N_2 & 0 & N_3 & 0 \\ 0 & N_1 & 0 & N_2 & 0 & N_3 \end{bmatrix}$$

形矩阵

$$u = N_1 u_1 + N_2 u_2 + N_3 u_3$$

$$v = N_1 v_1 + N_2 v_2 + N_3 v_3$$

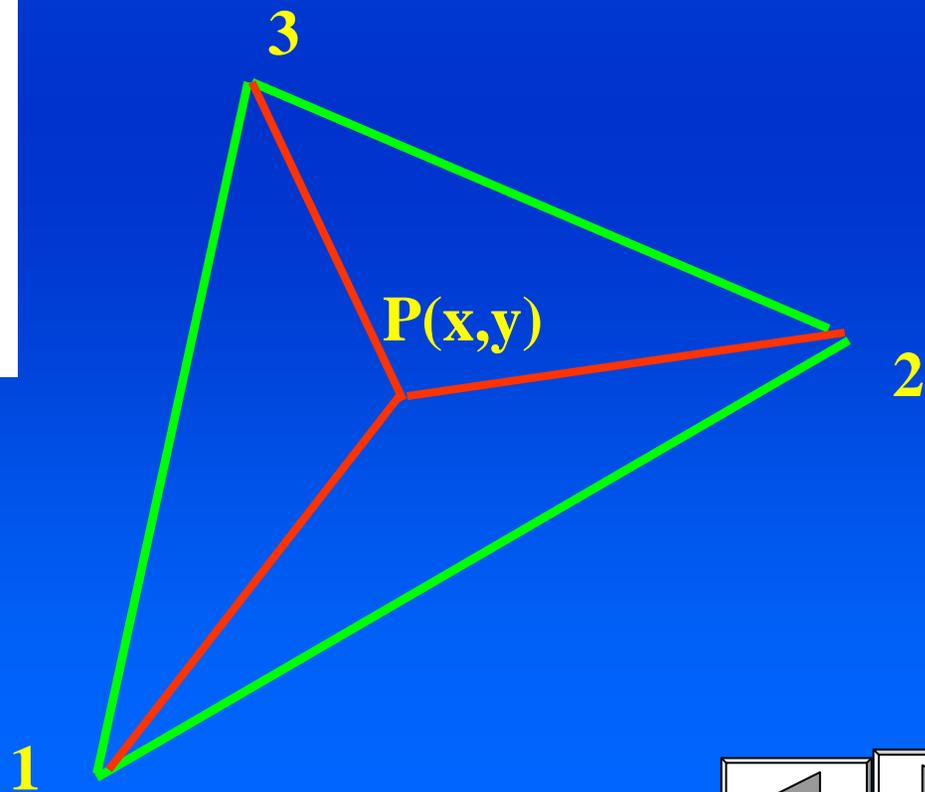


$$N_1 = \frac{1}{2\Delta} \begin{vmatrix} 1 & x & y \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix}$$

$$N_2 = \frac{1}{2\Delta} \begin{vmatrix} 1 & x & y \\ 1 & x_3 & y_3 \\ 1 & x_1 & y_1 \end{vmatrix}$$

$$N_3 = \frac{1}{2\Delta} \begin{vmatrix} 1 & x & y \\ 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \end{vmatrix}$$

形函数

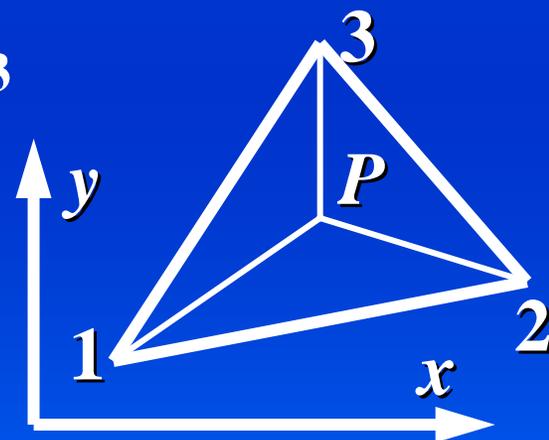


# 引入面积坐标

$$L_1 = \frac{1}{2\Delta} \begin{vmatrix} 1 & x & y \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix} = A_{P23}/A_{123}$$

$$L_2 = \frac{1}{2\Delta} \begin{vmatrix} 1 & x & y \\ 1 & x_3 & y_3 \\ 1 & x_1 & y_1 \end{vmatrix} = A_{P31}/A_{123}$$

$$L_3 = \frac{1}{2\Delta} \begin{vmatrix} 1 & x & y \\ 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \end{vmatrix} = A_{P12}/A_{123}$$



$$N_1=L_1, \quad N_2=L_2, \quad N_3=L_3$$

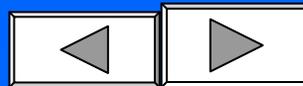
$$L_i = \frac{1}{2\Delta} (a_i + b_i x + c_i y)$$

$$i, j, k = 1, 2, 3$$

$$a_i = x_j y_k - y_j x_k$$

$$b_i = y_j - y_k$$

$$c_i = -x_j + x_k$$



## 4.4 解答的收敛性准则

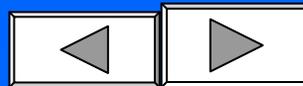
1) 位移模式（也称位移函数）必须包含刚体位移。

2) 位移模式必须包含常应变位移。

3) 位移模式必须保证单元间位移协调。

1)、2) 对平面问题也即要求具有常数项和坐标一次项，这称作“完备性准则”。

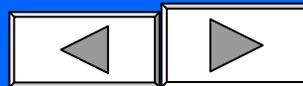
3) 称作“协调性准则”。既完备又协调的单元一定是收敛的。但不等于说非协调单元一定不收敛。



$$[N(x, y)] = \begin{bmatrix} N_1 & 0 & N_2 & 0 & N_3 & 0 \\ 0 & N_1 & 0 & N_2 & 0 & N_3 \end{bmatrix}$$

$$[N] = \left[ [N_1] \quad [N_2] \quad [N_3] \right]$$

$$N_i = L_i, i = 1, 2, 3$$



## 形函数的性质

$$1. \quad N_i = \begin{cases} 1 & x = x_i, y = y_i \\ 0 & x \neq x_i, y \neq y_i \end{cases}$$

$$2. \quad N_1 + N_2 + N_3 = 1$$

$$\{\mathbf{d}\} = [\mathbf{N}]\{\mathbf{a}\}$$



# 应变

$$\{\varepsilon\} = [A]^T \{d\}$$

$$\{\varepsilon\} = [A]^T \{d\} = [A]^T [N] \{a\}$$

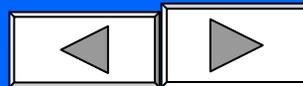
$$\{\varepsilon\} = [B] \{a\}$$

$$[B] = [A]^T [N]$$



$$[B] = \begin{bmatrix} \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix} \begin{bmatrix} N_1 & 0 & N_2 & 0 & N_3 & 0 \\ 0 & N_1 & 0 & N_2 & 0 & N_3 \end{bmatrix}$$

$$[B] = \frac{1}{2\Delta} \begin{bmatrix} b_1 & 0 & b_2 & 0 & b_3 & 0 \\ 0 & c_1 & 0 & c_2 & 0 & c_3 \\ c_1 & b_1 & c_2 & b_2 & c_3 & b_3 \end{bmatrix}$$



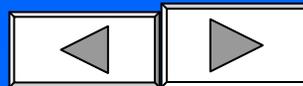
# 应力

$$\{\sigma\} = [D]\{\varepsilon\}$$

$$[D] = \frac{E}{1-\mu^2} \begin{bmatrix} 1 & \mu & 0 \\ \mu & 1 & 0 \\ 0 & 0 & \frac{1-\mu}{2} \end{bmatrix}$$

平面应力

$$\{\sigma\} = [D][B]\{a\}$$



## 结点力

由虚功原理:

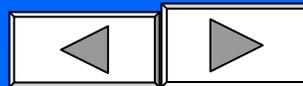
在一个平衡状态  $(\mathbf{u}, \boldsymbol{\varepsilon}, \boldsymbol{\sigma})$ ,

设有虚位移  $\{\mathbf{a}^*\}$ , 对应的应变为  $\{\boldsymbol{\varepsilon}^*\} = [\mathbf{B}]\{\mathbf{a}^*\}$ ,

则:

$$\begin{aligned}\{\mathbf{a}^*\}^T \{\mathbf{F}\} &= \iiint_{V_e} \{\boldsymbol{\varepsilon}^*\}^T \{\boldsymbol{\sigma}\} dV \\ &= \iiint_{V_e} \{\mathbf{a}^*\}^T [\mathbf{B}]^T \{\boldsymbol{\sigma}\} dV\end{aligned}$$

$$\begin{aligned}\{\mathbf{F}\} &= \iiint_{V_e} [\mathbf{B}]^T \{\boldsymbol{\sigma}\} dV \\ &= \iint_{S_e} [\mathbf{B}]^T \{\boldsymbol{\sigma}\} t dS\end{aligned}$$



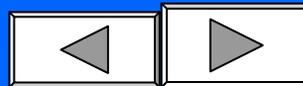
$$\begin{aligned}\{F\} &= \iint_{S_e} [B]^T \{\sigma\} t dS \\ &= \iint_{S_e} [B]^T [D][B]\{a\} t dS \\ &= [B]^T [D][B] t \Delta \{a\}\end{aligned}$$

$$\{F\} = [k]\{a\}$$

$$[k] = [B]^T [D][B] t \Delta$$

单元刚度阵

有限元方程



作业：  
计算单元刚度阵  
P167  
7-1,  
7-3



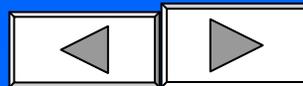
## 4.5 单元刚度阵的另一推导

总势能:

$$\Pi = \frac{1}{2} \iiint_V \{\sigma\}^T \{\varepsilon\} dV - \iiint_V \{T\}^T \{u\} dV - \iint_{S_1} \{\bar{T}\}^T \{u\} dS$$

不计体力:

$$\Pi = \frac{1}{2} \{a\}^T [B]^T [D][B] \{a\} t \Delta - \{a\}^T \{F\}$$



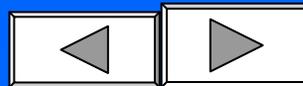
由最小势能原理:

$$\frac{\partial \Pi}{\partial \{a\}} = 0$$

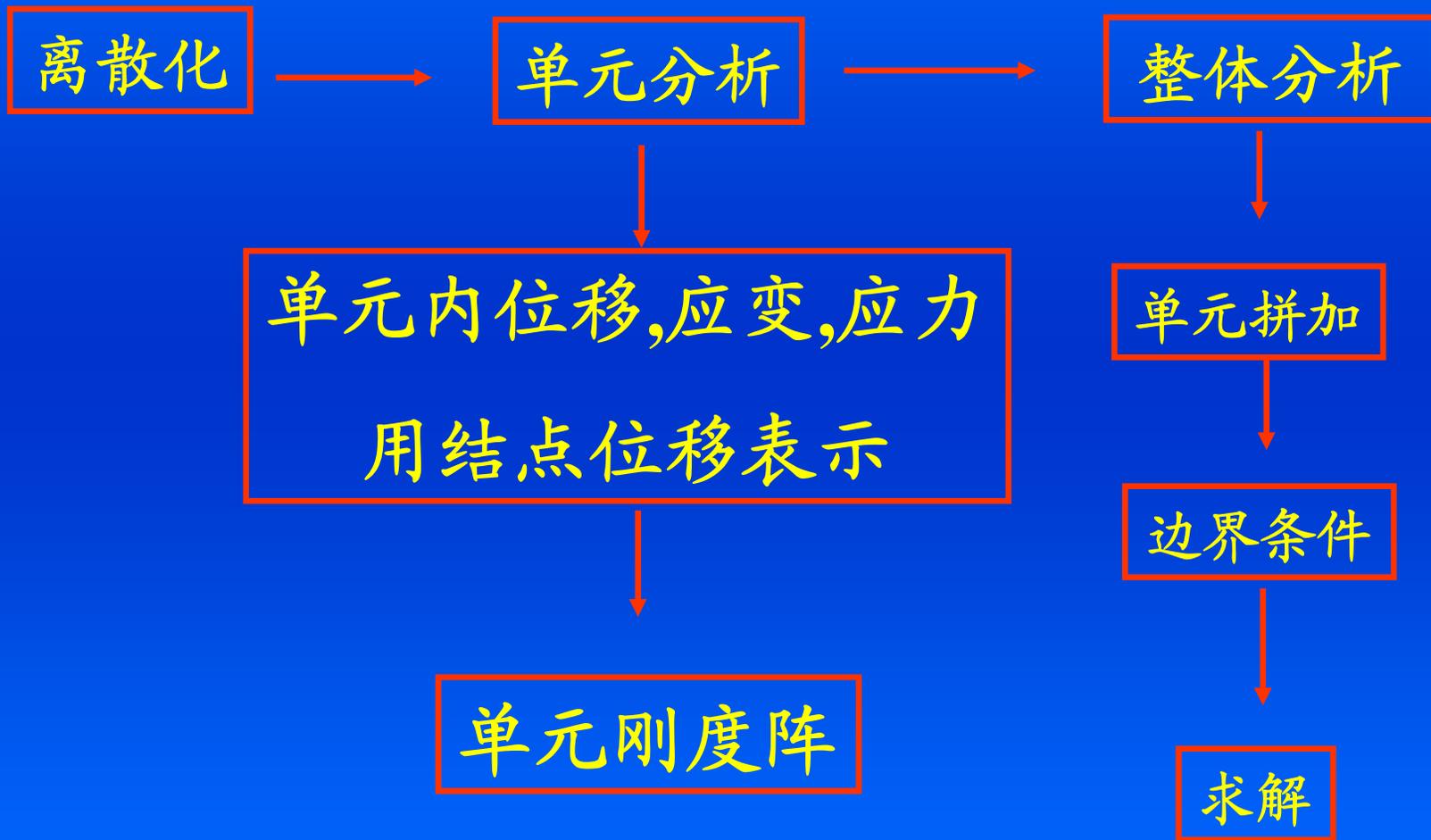
$$[B]^T [D][B]\{a\}t\Delta - \{F\} = 0$$

$$[k]\{a\} = \{F\}$$

$$[k] = [B]^T [D][B]t\Delta$$



## 4.6 整体分析



# 整体分析的步骤

建立整体  
刚度矩阵



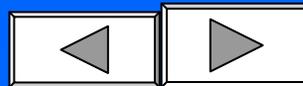
引入支  
承条件



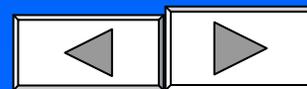
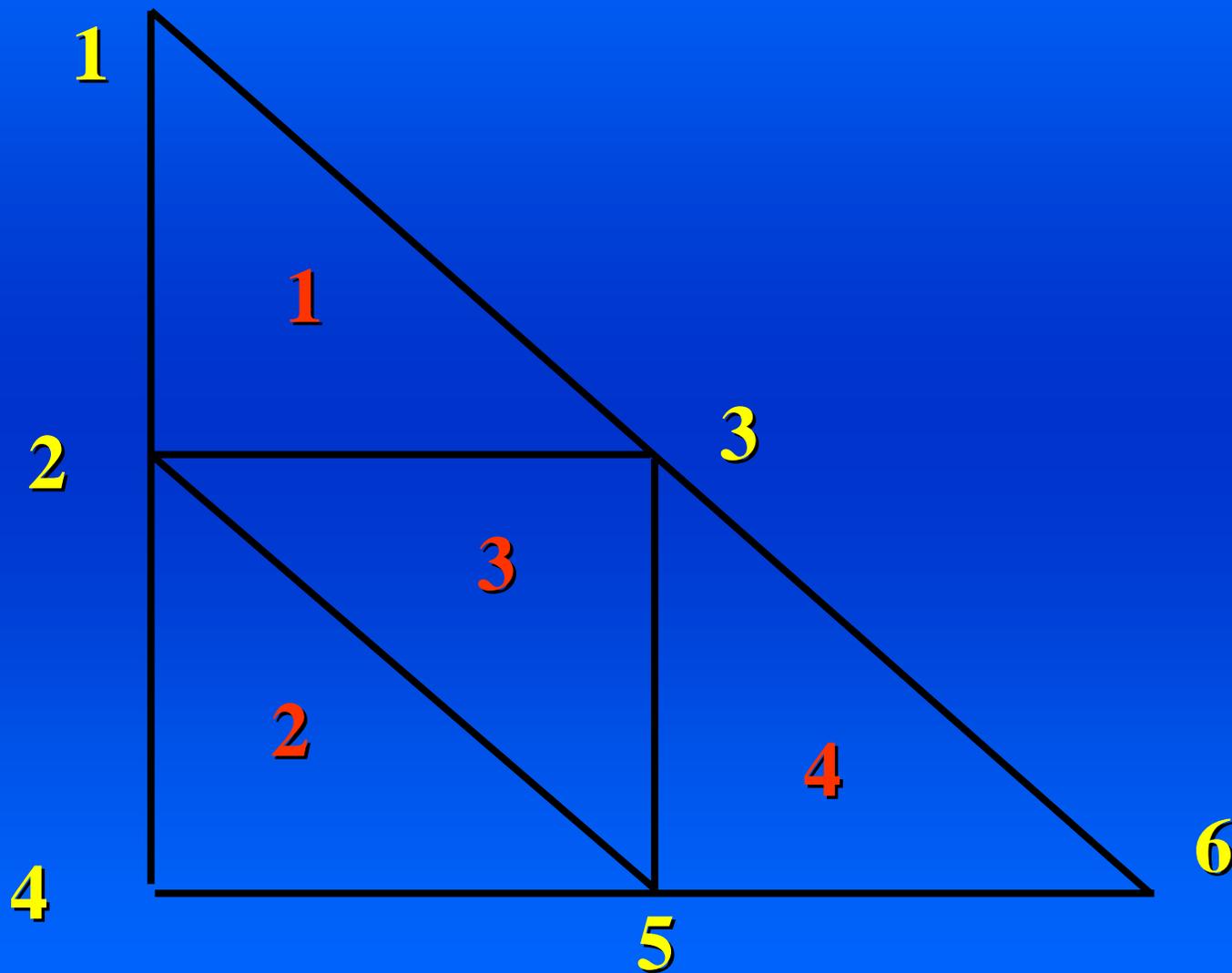
解方程  
求位移



求应力



# 总体刚度阵的形成:

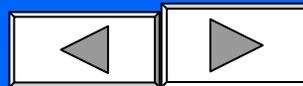


$$\begin{bmatrix} x & x & x & x & x & x \\ x & x & x & x & x & x \\ x & x & x & x & x & x \\ x & x & x & x & x & x \\ x & x & x & x & x & x \\ x & x & x & x & x & x \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix} = \begin{Bmatrix} U_1 \\ V_1 \\ U_2 \\ V_2 \\ U_3 \\ V_3 \end{Bmatrix}, \quad [k]_1 \{a\}_1 = \{F\}_1$$

$$\begin{bmatrix} x & x & x & x & x & x & .. & 0 \\ x & x & x & x & x & x & .. & 0 \\ x & x & x & x & x & x & .. & 0 \\ x & x & x & x & x & x & .. & 0 \\ x & x & x & x & x & x & .. & 0 \\ x & x & x & x & x & x & .. & 0 \\ .. & .. & .. & .. & .. & .. & .. & .. \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \\ u_4 \\ v_6 \end{Bmatrix} = \begin{Bmatrix} U_1 \\ V_1 \\ U_2 \\ V_2 \\ U_3 \\ V_3 \\ .. \\ 0 \end{Bmatrix}, \quad [K]_1 \{a\} = \{\hat{F}\}_1$$

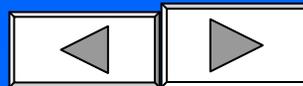


$$\begin{bmatrix} y & y & y & y & y & y \\ y & y & y & y & y & y \\ y & y & y & y & y & y \\ y & y & y & y & y & y \\ y & y & y & y & y & y \\ y & y & y & y & y & y \end{bmatrix} \begin{Bmatrix} u_2 \\ v_2 \\ u_4 \\ v_4 \\ u_5 \\ v_5 \end{Bmatrix} = \begin{Bmatrix} U_2 \\ V_2 \\ U_4 \\ V_4 \\ U_5 \\ V_5 \end{Bmatrix}, \quad [k]_2 \{a\}_2 = \{F\}_2$$



$$\begin{bmatrix}
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & y & y & 0 & 0 & y & y & y & y & 0 & 0 \\
 0 & 0 & y & y & 0 & 0 & y & y & y & y & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & y & y & 0 & 0 & y & y & y & y & 0 & 0 \\
 0 & 0 & y & y & 0 & 0 & y & y & y & y & 0 & 0 \\
 0 & 0 & y & y & 0 & 0 & y & y & y & y & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0
 \end{bmatrix}
 \begin{Bmatrix}
 u_1 \\
 v_1 \\
 u_2 \\
 v_2 \\
 u_3 \\
 v_3 \\
 u_4 \\
 v_4 \\
 u_5 \\
 v_5 \\
 u_6 \\
 v_6
 \end{Bmatrix}
 =
 \begin{Bmatrix}
 0 \\
 0 \\
 U_2 \\
 V_2 \\
 0 \\
 0 \\
 U_4 \\
 V_4 \\
 U_5 \\
 V_5 \\
 0 \\
 0
 \end{Bmatrix}$$

$$[K]_2 \{a\} = \{\hat{F}\}_2$$



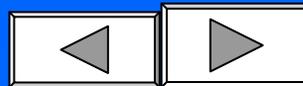
## 总刚和总荷载:

$$[K] = [K]_1 + [K]_2 + \dots + [K]_4$$

$$\{F\} = \{\hat{F}\}_1 + \{\hat{F}\}_2 + \dots + \{\hat{F}\}_4$$

## 最后的有限元方程:

$$[K]\{a\} = \{F\}$$

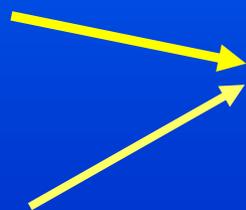


# 总刚的性质:

1.对称性

2.稀疏性

3.带状性



与结点编号有关

4.主元大于0

$k$	$k$	$k$	$k$	$k$	$k$	0	0	0	0	0	0
	$k$	$k$	$k$	$k$	$k$	0	0	0	0	0	0
		$k$	0	0							
			$k$	0	0						
				$k$	$k$	0	0	$k$	$k$	$k$	$k$
					$k$	0	0	$k$	$k$	$k$	$k$
						$k$	$k$	$k$	$k$	0	0
							$k$	$k$	$k$	0	0
								$k$	$k$	$k$	$k$
									$k$	$k$	$k$
										$k$	$k$
											$k$

- 1
- 2
- 3
- 4
- 5
- 6
- 7
- 8
- 9
- 10
- 11
- 12

1

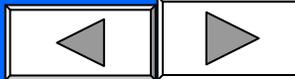
2

3

4

5

6



$k$	$k$	0	0	$k$	$k$	0	0	0	0	$k$	$k$	1
	$k$	0	0	$k$	$k$	0	0	0	0	$k$	$k$	2
		$k$	$k$	$k$	$k$	0	0	$k$	$k$	0	0	3
			$k$	$k$	$k$	0	0	$k$	$k$	0	0	4
				$k$	$k$	0	0	$k$	$k$	$k$	$k$	5
					$k$	0	0	$k$	$k$	$k$	$k$	6
						$k$	$k$	$k$	$k$	$k$	$k$	7
							$k$	$k$	$k$	$k$	$k$	8
								$k$	$k$	0	0	9
									$k$	0	0	10
										$k$	$k$	11
											$k$	12

1

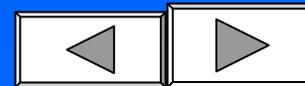
6

4

3

5

2



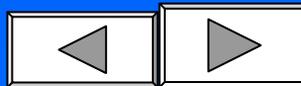
## 4.7 边界条件的引入

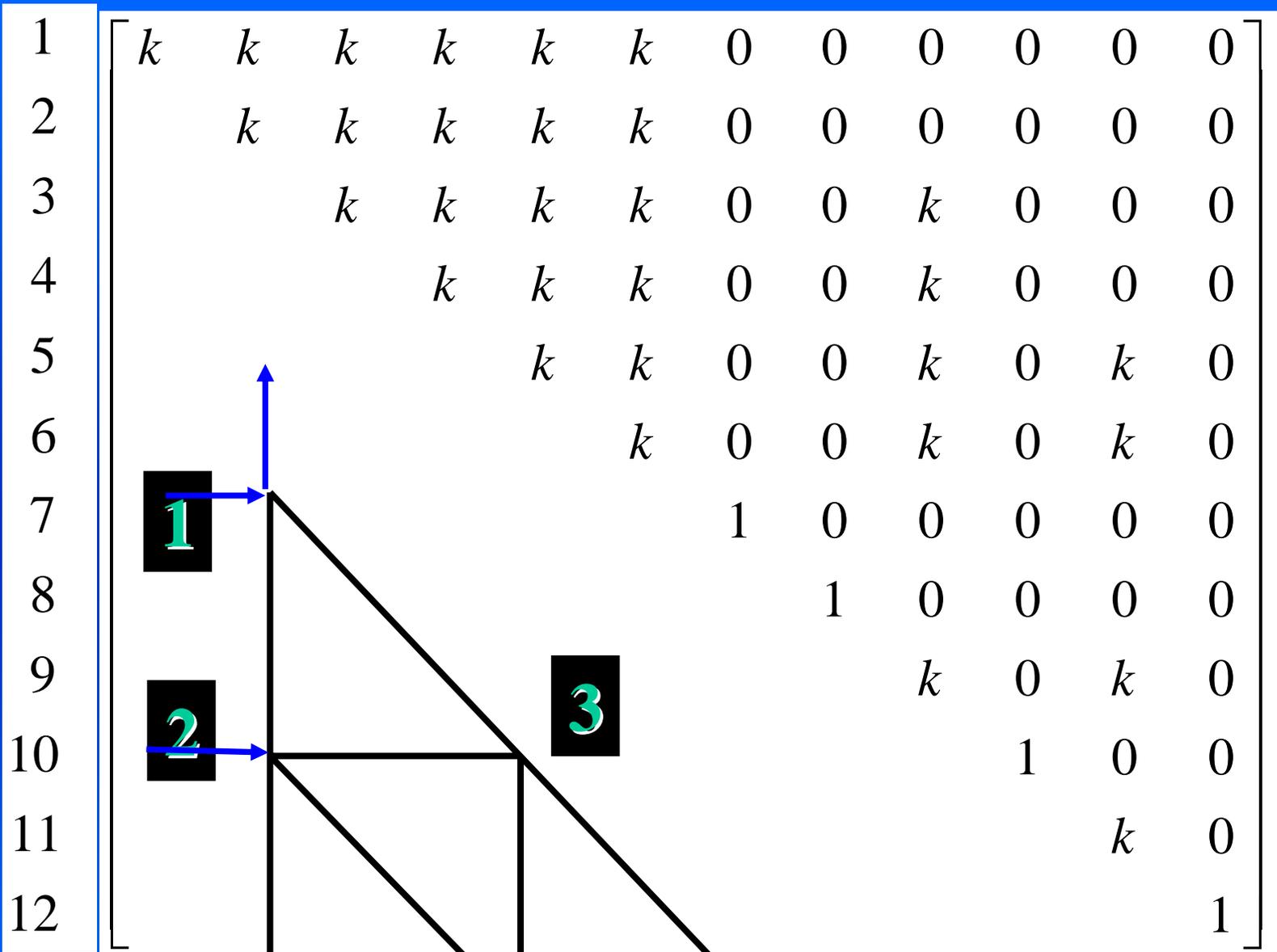
### 1. 总刚的修改:

受约束的自由度对应的主元=1,  
该行和列的其他元素=0.

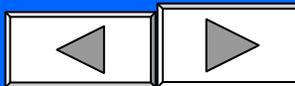
### 2. 荷载列阵的修改:

受约束自由度的荷载项=0

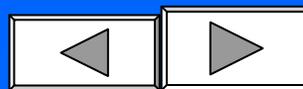




$$\begin{Bmatrix} p_{x1} \\ p_{y1} \\ p_{x2} \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{Bmatrix}$$

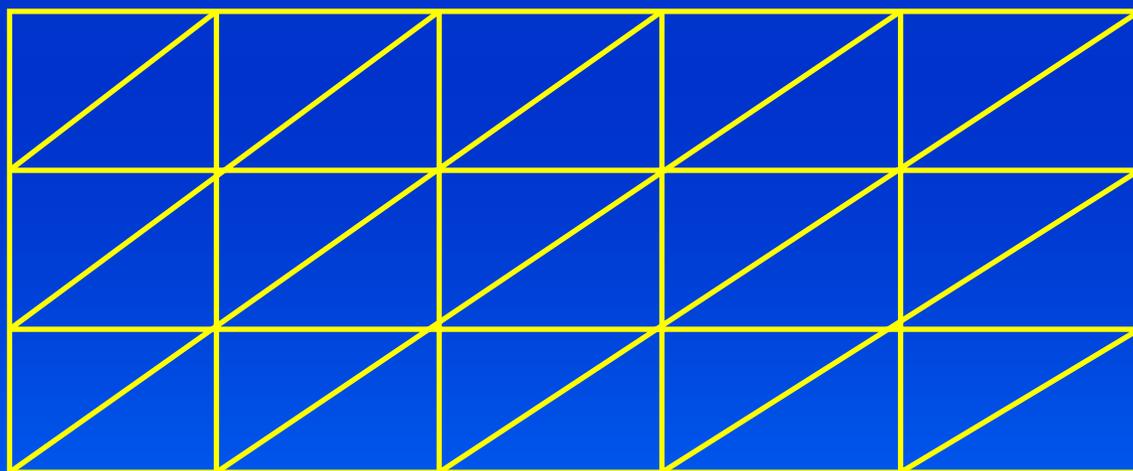


作业：  
计算总体刚度阵  
P167  
7-5，  
求解位移  
7-4



## 补充作业1:

1. 对下列有限元划分, 进行合理的最佳节点编号和单元编号



## 补充作业2:

2. 对下列有限元划分, 进行合理的最佳  
节点编号和单元编号

